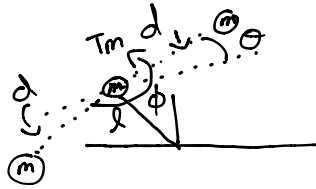


Apply Lagrange's Equations with generalized Forces to develop a model for an idealized tight-rope walker



T_m : torque applied by tight rope walker to the balance beam

generalized coordinates of tight-rope walker

$\rightarrow \phi$: lean angle relative to vertical

$\rightarrow \theta$: rotation angle of the balance beam relative to walkers hips

$L = T - V$ T : total KE, V : total PE

$$\frac{d}{dt} \frac{\partial L}{\partial \dot{\phi}} - \frac{\partial L}{\partial \phi} = 0$$

$$\frac{d}{dt} \frac{\partial L}{\partial \dot{\theta}} - \frac{\partial L}{\partial \theta} = 0 \quad \text{if } T_m = 0$$

If $T_m \neq 0$, a non-conservative force

$$\frac{d}{dt} \frac{\partial L}{\partial \dot{\phi}} - \frac{\partial L}{\partial \phi} = Q_\phi$$

$$\frac{d}{dt} \frac{\partial L}{\partial \dot{\theta}} - \frac{\partial L}{\partial \theta} = Q_\theta$$

$\begin{bmatrix} Q_\phi \\ Q_\theta \end{bmatrix} \rightarrow$ generalized force vector

Corresponding to T_m

\rightarrow generalized coordinate space $\begin{bmatrix} \phi \\ \theta \end{bmatrix}$

determine Q_ϕ by determining virtual work done by virtual displacement $d\phi$

how much work happens when we tweak ϕ and freeze θ

definition: $\delta W_\phi = Q_\phi \delta \phi = 0$ in this case

$$\text{therefore: } Q_\phi = \frac{0}{d\phi} = 0$$

determine Q_θ using virtual work

$$\delta W_\theta = Q_\theta \delta \theta = T_m \delta \theta$$

$$\text{so } Q_\theta = T_m \quad (\text{Lagrange} = T_m)$$

Get Cartesian position coordinates of each mass as functions of ϕ, θ

differentiate to get the velocities, then $T = \frac{1}{2} m V^2$, $V = mgh$

$$x_m = -l \sin \phi \quad y_m = l \cos \phi \quad \dot{x}_m = -l \dot{\phi} \cos \phi \quad \dot{y}_m = -l \dot{\phi} \sin \phi$$

$$V_a = mgy_m = mgl \cos \phi \quad T_a = \frac{1}{2} m (\dot{x}_m^2 + \dot{y}_m^2) = \frac{1}{2} m l^2 \dot{\phi}^2$$

now for right most mass \triangle :

$$x_{mr} = -l \sin \phi + d \cos(\phi + \theta) \quad y_{mr} = -l \cos \phi + d \sin(\phi + \theta)$$

$$\dot{x}_{mr} = -l \dot{\phi} \cos \phi - d(\dot{\theta} + \dot{\phi}) \sin(\phi + \theta) \quad \dot{y}_{mr} = -l \dot{\phi} \sin \phi + d(\dot{\phi} + \dot{\theta}) \cos(\phi + \theta)$$

$$V_b = Mgy_{mr} = Mg[l \cos \phi + d \sin(\phi + \theta)]$$

$$T_b = \frac{1}{2} M (\dot{x}_{mr}^2 + \dot{y}_{mr}^2) = \frac{1}{2} M [l^2 \dot{\phi}^2 + 2ld \dot{\phi}(\dot{\phi} + \dot{\theta}) \sin \theta + d^2(\dot{\phi} + \dot{\theta})^2]$$

Left-most mass: d is opposite

$$V_c = Mg[l \cos \phi - d \sin(\phi + \theta)]$$

$$T_c = \frac{1}{2} M [l^2 \dot{\phi}^2 - 2ld \dot{\phi}(\dot{\phi} + \dot{\theta}) \sin \theta + d^2(\dot{\phi} + \dot{\theta})^2]$$

Get total KE's, total PE's

$$T_{tot} = T_a + T_b + T_c = \frac{1}{2}(m + 2M) l^2 \dot{\phi}^2 + M d^2 (\dot{\phi} + \dot{\theta})^2$$

$$V_{tot} = V_a + V_b + V_c = (m + 2M) l g \cos \phi$$

$$\frac{d}{dt} \left[\frac{\partial T}{\partial \dot{\phi}} \right] - \frac{\partial T}{\partial \phi} + \frac{\partial V}{\partial \phi} = 0 \quad V \text{ does not depend on } \dot{\phi}$$

$$\frac{d}{dt} [(m + 2M) l^2 \dot{\phi} + 2M d^2 (\dot{\phi} + \dot{\theta})] - [m + 2M] l g \sin \phi = 0$$

$$[(m + 2M) l^2 + 2M d^2] \ddot{\phi} + 2M d^2 \ddot{\theta} - [m + 2M] l g \sin \phi = 0 \quad (1)$$

$$\frac{d}{dt} \left[\frac{\partial T}{\partial \dot{\theta}} \right] - \frac{\partial T}{\partial \theta} + \frac{\partial V}{\partial \theta} = T_m \rightarrow \frac{d}{dt} [2M d^2 (\dot{\phi} + \dot{\theta})] = T_m$$

$$[2M d^2] \ddot{\phi} + [2M d^2] \ddot{\theta} = T_m \quad (2)$$

solve for $\ddot{\phi}$ and $\ddot{\theta}$

$$\ddot{\phi} = \frac{g}{l} \sin \theta - \left[\frac{1}{(m + 2M) l^2} \right] T_m \quad \ddot{\theta} = -\frac{g}{l} \sin \phi + \left[\frac{m l^2 + 2M (l^2 + d^2)}{2M d^2 (m + 2M) l^2} \right] T_m$$

does $\dot{E} = T_m \dot{\theta}$?

$$\dot{E} = \frac{dT}{d\dot{\phi}} \ddot{\phi} + \frac{\partial T}{\partial \dot{\theta}} \ddot{\theta} + \frac{\partial V}{\partial \dot{\phi}} \dot{\phi}$$

$$\frac{dT}{d\phi} \left[\frac{g}{2} \sin \theta - (\dots) T_m \right] + \frac{dT}{d\theta} \left[-\frac{g}{2} \sin \phi + (\dots) T_m \right] + \frac{\partial V}{\partial \phi} \dot{\phi} \dots = T_m \dot{\theta}$$

if you grab the balance beam: $\theta=0, \dot{\phi}=0, \ddot{\theta}=0$

$$0 = -\frac{g}{2} \sin \phi + [\dots] T_m \quad \text{solve for } T_m \text{ and put in } \ddot{\phi} \text{ equation}$$

$$\ddot{\phi} = \left[\frac{(m+2M)g}{(m+2M)l^2 + 2Md^2} \right] \frac{g}{2} \sin \phi$$